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Industriestraße 3
77815 Bühl

0790

### Patentansprüche

- 1. Kraftfahrzeug mit einem Antriebsstrang, mit einer Brennkraftmaschine und einem nachgeschalteten Getriebe, wobei zwischen einer Kurbelwelle der Brennkraftmaschine und einer Eingangswelle des Getriebes ein geteiltes Schwungrad mit zumindest zwei gegeneinander entgegen der Wirkung einer Energiespeichervorrichtung relativ verdrehbaren Massen angeordnet ist und zumindest eine der Massen mit der Kurbelwelle und zumindest eine Masse mit einer Eingangswelle des Getriebes unter Zwischenschaltung einer Reibungskupplung verbindbar ist.
- 2. Kraftfahrzeug mit einem Antriebsstrang, mit einer Brennkraftmaschine und einem Getriebe, wobei zwischen einer Kurbelwelle der Brennkraftmaschine und einer Eingangswelle des Getriebes ein hydrodynamischer Drehmomentwandler angeordnet ist und dieser hydrodynamische Drehmomentwandler mittels einer Wandlerüberbrückungskupplung überbrückbar ist.
- 3. Kraftfahrzeug mit einem Antriebsstrang, mit einer Brennkraftmaschine und einem Getriebe, wobei zwischen einer Kurbelwelle der Brennkraftmaschine und zumindest einer Eingangswelle des Getriebes zumindest eine Reibungskupplung vorgesehen ist, die mittels eines hydraulischen Ausrücksystems, zumindest bestehend aus einem hydraulischen Nehmerzylinder, einem hydraulischen Geberzylinder und einer diese verbindenden hydraulischen Leitung, betätigt wird.
- 4. Kraftfahrzeug mit einem Antriebsstrang, mit einer Brennkraftmaschine und einem Getriebe, sowie einer eine Kurbelwelle der Brennkraftmaschine und eine Getriebeeingangswelle des Getriebes verbindende Drehmomentübertragungseinrichtung, wie Reibungskupplung oder hydrodynamischer Drehmomentwandler, wobei in zumindest eine Wange der Kurbelwelle ein Schwingungstilger zur Verminderung von Drehungleichförmigkeiten der Brennkraftmaschine vorgesehen ist.



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5. Kraftfahrzeug mit einem Antriebsstrang, mit einer Brennkraftmaschine und einem bezüglich seiner Übersetzung kontinuierlich verstellbaren Umschlingungsmittelgetriebe, bestehend aus einem Getriebeeingangsteil mit einem ersten Kegelscheibensatz und einem Getriebeausgangsteil, mit einem zweiten Kegelscheibensatz sowie einem endlosen, beide Kegelscheibensätze umschlingenden Umschlingungsmittel, wobei jeweils eine Kegelscheibe eines Kegelscheibensatzes gegenüber einer zweiten Scheibe des Kegelscheibensatzes axial mittels hydraulischen Drucks verlagerbar ist.

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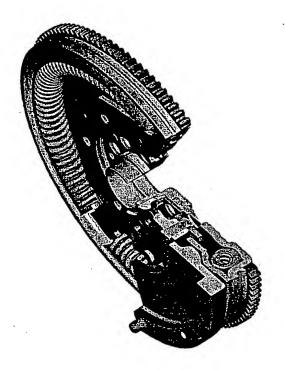
25

- Kraftfahrzeug mit einem Antriebsstrang, insbesondere nach Anspruch 5, dadurch gekennzeichnet, dass das Umschlingungsmittel aus einer Kette gebildet ist, die aus in
  Laufrichtung hintereinander folgenden Laschenverbänden besteht, die quer zur Laufrichtung von Stegen durchdrungen werden, wobei diese Stege an ihren axialen Enden jeweils in Reibeingriff mit den beiden Kegelscheiben eines Kegelscheibenpaares
  stehen.
  - 7. Kraftfahrzeug mit einem Antriebsstrang, insbesondere nach Anspruch 5, dadurch gekennzeichnet, dass das kontinuierlich verstellbare Umschlingungsmittelgetriebe Teil eines leistungsverzweigten Getriebes ist.
  - 8. Kraftfahrzeug mit einem Antriebsstrang, mit einer Brennkraftmaschine und einem Getriebe, wobei das Getriebe ein Kurbelgetriebe ist.
  - 9. Kraftfahrzeug mit einem Antriebsstrang, einer Brennkraftmaschine und einem Getriebe, wobei das Getriebe ein automatisiertes Schaltgetriebe mit zumindest einer Getriebeeingangswelle ist und die Schaltvorgänge mittels einer Aktorik erfolgen, die zumindest aus einem Elektromotor gebildet ist.
- 10. Kraftfahrzeug mit einem Antriebsstrang, mit einer Brennkraftmaschine mit einer Kur-30 belwelle und einem Getriebe mit zumindest einer Getriebeeingangswelle, wobei eine Elektromaschine mit der Kurbelwelle und/oder der Getriebeeingangswelle koppelbar ist.

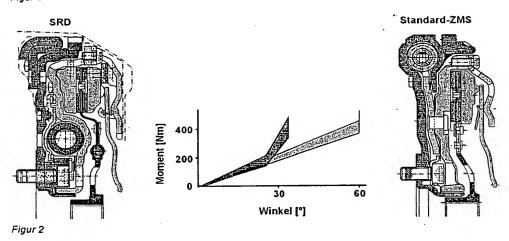
11. Kraftfahrzeug mit einer Pumpe zur hydraulischen Versorgung von Lenkungssystemen und/oder Fahrwerkssystemen und/oder Getriebesystemen, insbesondere Flügelzellenpumpe oder Radialkolbenpumpe oder Zahnradpumpe, wobei die Pumpe Einrichtungen zur Regelung des Volumenstroms aufweist.

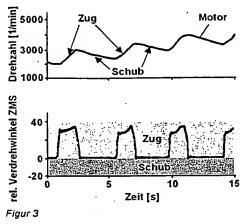
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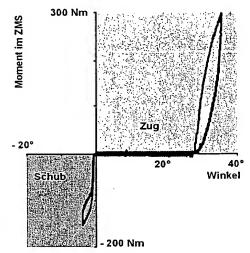
- 12. Kraftfahrzeug mit einer Pumpe, wie Flügelzellenpumpe oder G-Rotor-Pumpe oder Rollenzellenpumpe, insbesondere zur Schmierölversorgung eines Verbrennungsmotors, wobei der Volumenstrom der Pumpe temperaturabhängig beeinflussbar ist.
- 13. Kraftfahrzeug mit einer Klimaanlage mit Klimakompressor, wie Schwenkringkompressor, insbesondere zur Verwendung von CO<sub>2</sub> als Kältemittel, wobei die Klimaanlage zur Kühlung und/oder Erwärmung durch Verwendung als Wärmepumpe einsetzbar ist.
  - 15 14. Kraftfahrzeug mit einem Getriebe, insbesondere automatisiertes Schaltgetriebe, wobei verschiedene Schalt- und Kupplungsstrategien hydraulisch und/oder elektromotorisch ausgeführt werden können.



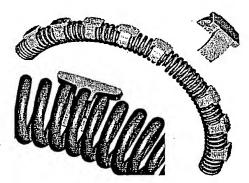
Figur 1



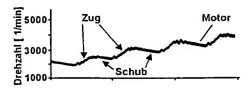


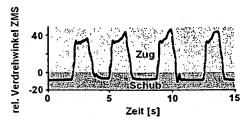


Figur 4

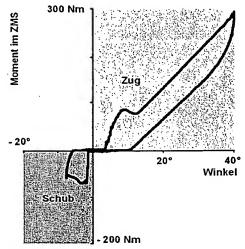


Figur 5

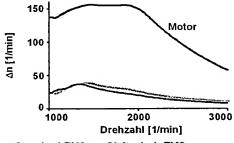




Figur 6

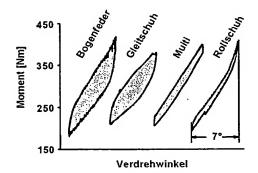


Figur 7



Standard-ZMS = Gleitschuh-ZMS

Figur 8



Figur 9

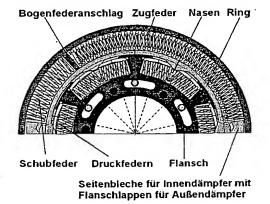


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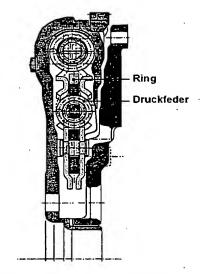
- keine vorgespannte Bogenfeder
- M<sub>Anschlag</sub> < M<sub>Motor</sub>

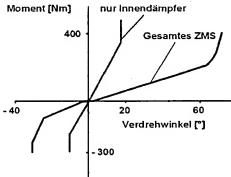
#### Innendämpfer:

- reibungsfrei
- Serienschaltung 2er Druckfedern
- M<sub>Anschlag</sub> < M<sub>Motor</sub>

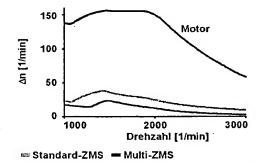


Figur 10

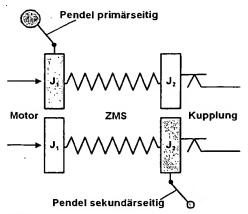




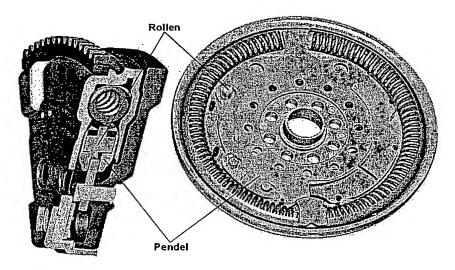
Figur 11



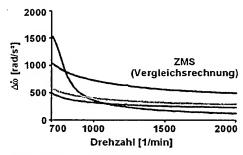
Figur 12



Figur 13



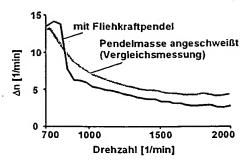
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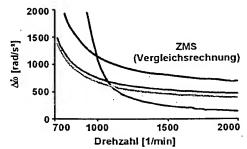
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Figur 15



Figur 16

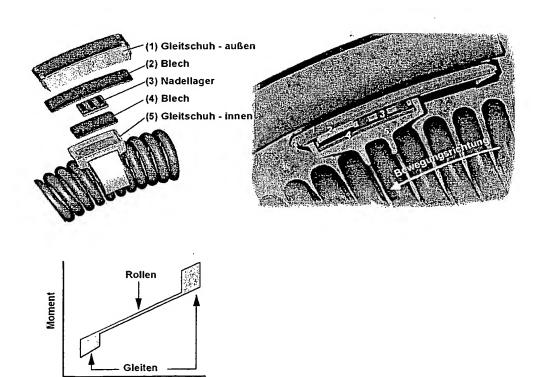


Eigenfrequenz abgestimmt auf:

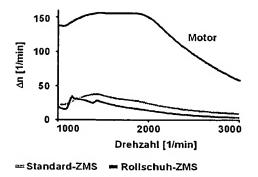
=2. Ordnung ==2,5. Ordnung

Winkel

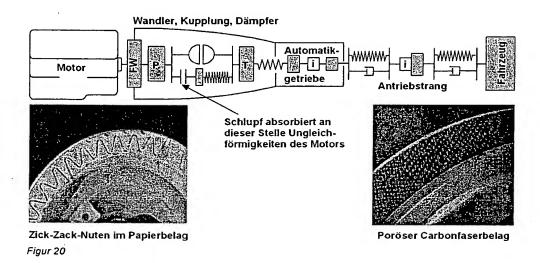
Figur 17



Figur 18



Figur 19



#### **Genuteter Deckel**



#### Ungenuteter Reibbelag



Figur 21



Kolbenblech (Stahl)

Wärmeübergang in den Deckel und dann in das Öl der sich vorbeibewegenden Nuten

#### Genuteter Deckel (Stahl)



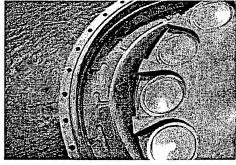
Kolbenblech (Stahl)

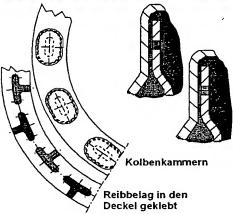
Mehr Nuten bedeutet besserer Wärmeübergang bei gleichbleibender Durchflußrate

🜃 Ölnuten

Reibbelag (Wärmeisolator)

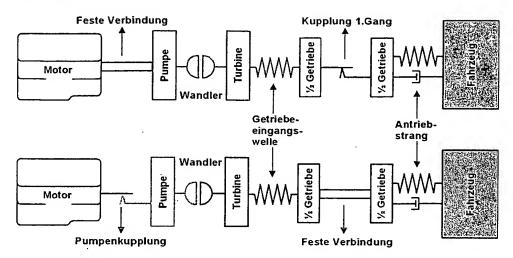
Figur 22





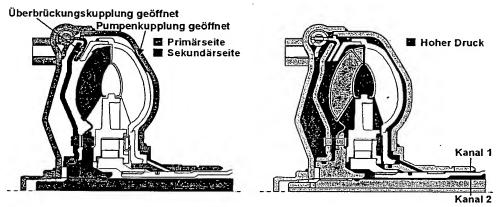
Figur 23

#### Schaltvorgang von Leerlauf nach "D"

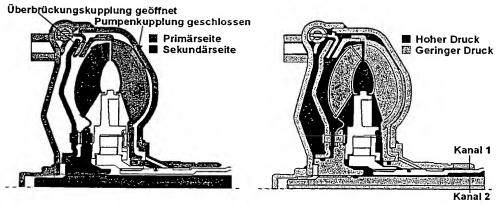


#### Betätigung der Kupplung zwischen Deckel und Pumpe

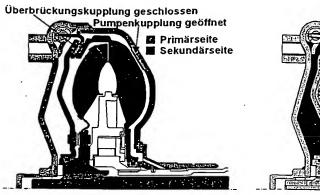
Figur 24



Figur 25



Figur 26

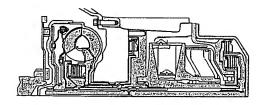


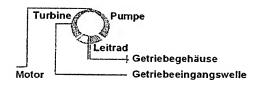
Hoher Druck
Geringer Druck

Kanal 1

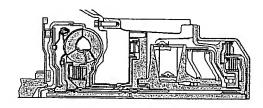
Kanal 2

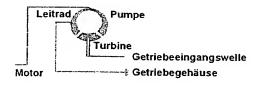
Figur 27



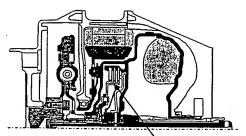


Figur 28

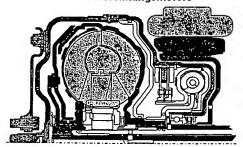


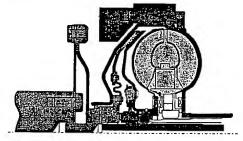


Figur 29



Kupplung zur Abkopplung des Verbrennungsmotors

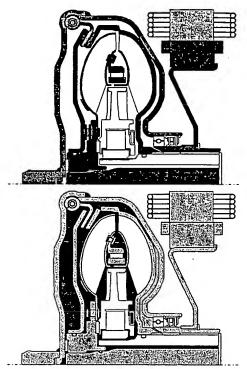




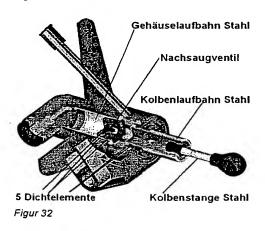
■ E-Maschine

■ Wandler

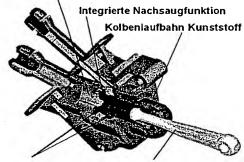
Figur 30



Figur 31



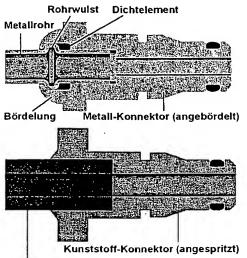
#### Gehäuselaufbahn Kunststoff



2 Dichtelemente

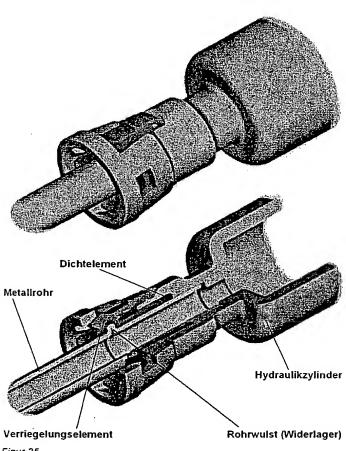
mente Kolbenstange Kunststoff

Figur 33

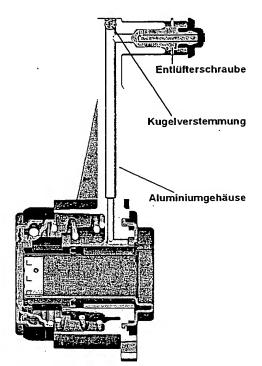


Kunststoffrohr

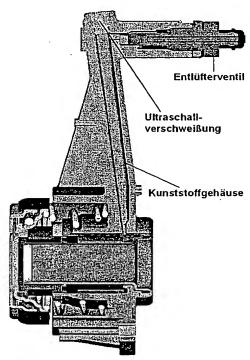
Figur 34



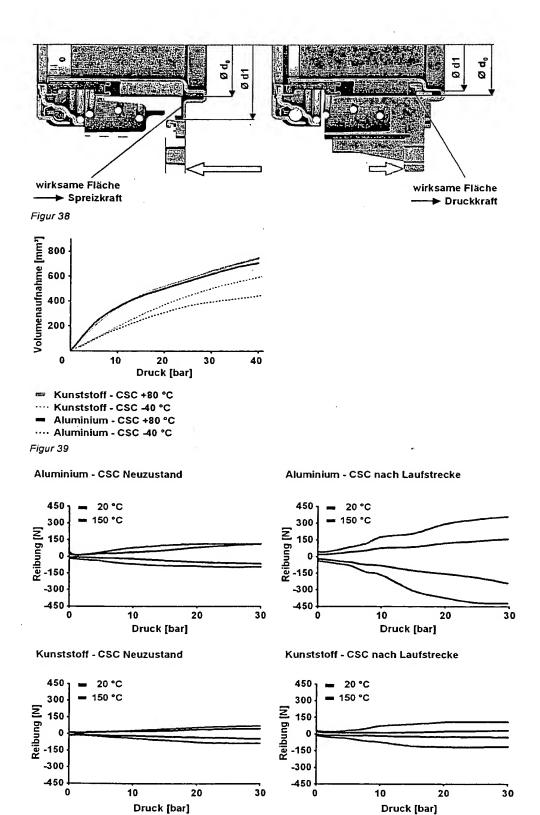
Figur 35



Figur 36



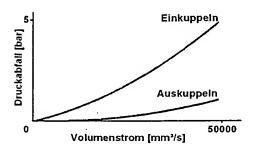
Figur 37

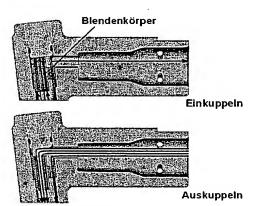


Figur 40

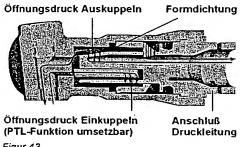


Figur 41

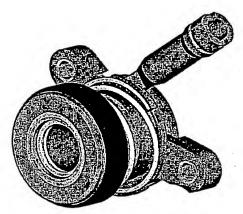




Figur 42



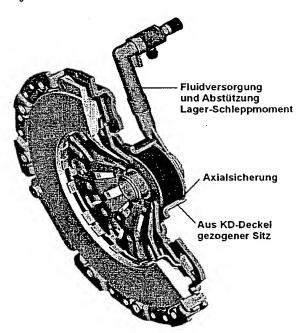
Figur 43



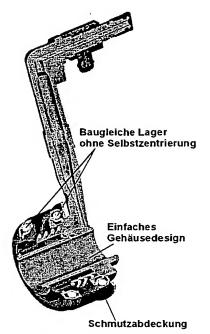
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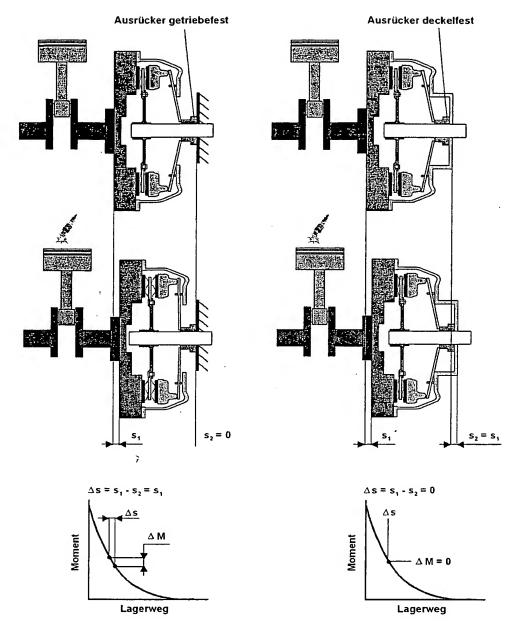
Figur 45



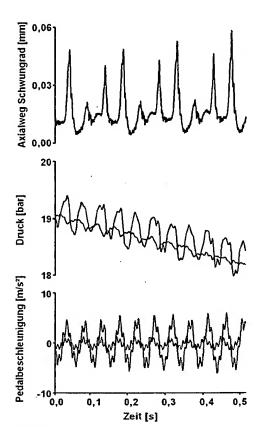
Figur 46



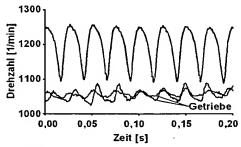
Figur 47



Figur 48

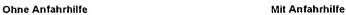


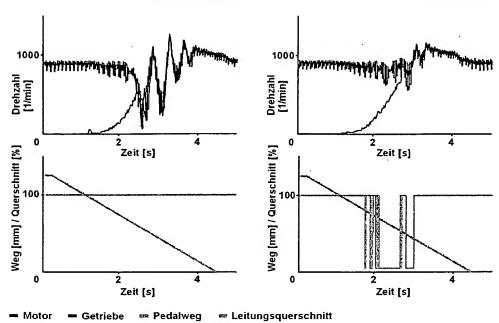
Figur 49



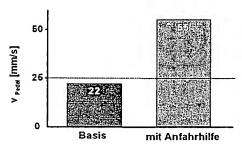
- MotorAusrücker getriebefestAusrücker deckelfest

Figur 50

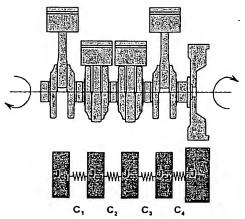




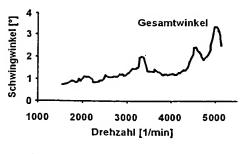
Figur 51

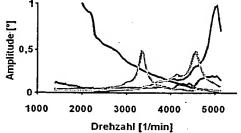


Figur 52



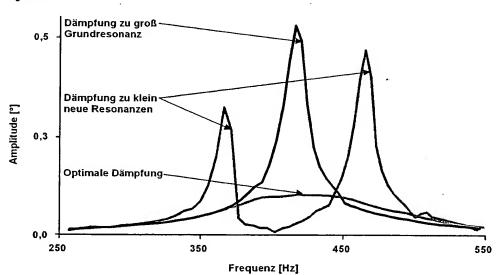
Figur 53





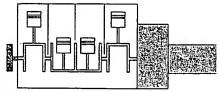
- = 2,5.Ordnung = 5,0.Ordnung ≈ 7,5.Ordnung = 4,5.Ordnung ≈ 5,5.Ordnung

Figur 54

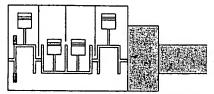




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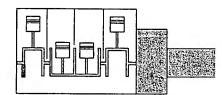


aussenliegender Dämpfer



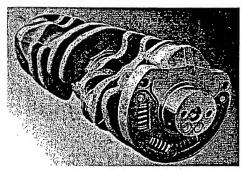
interner Dämpfer deutsches Patent 536 929





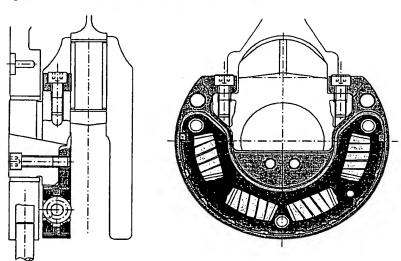
interner Dämpfer ICD

Figur 56

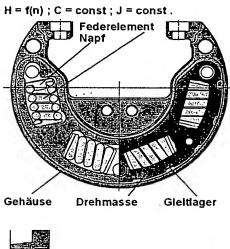


Figur 57



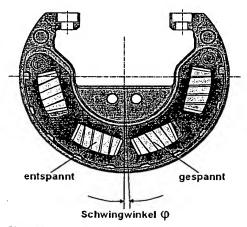


Figur 58



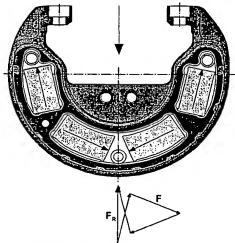


Figur 59



Figur 60





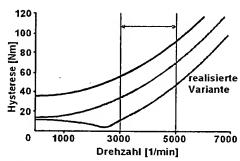
F = Federvorspannung

F<sub>R</sub> = Resultierende der Federvorspannung

Fliehkraft

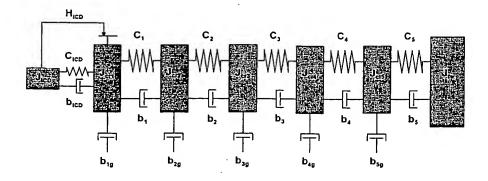
Figur 61

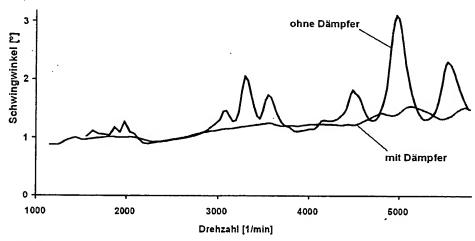
# Hauptarbeitsbereich



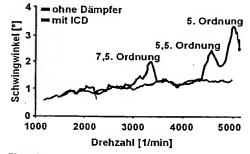
Figur 62





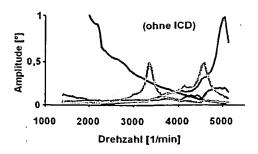


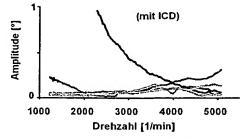
Figur 63



Figur 64

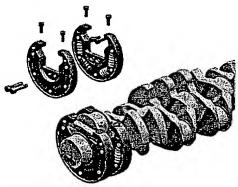




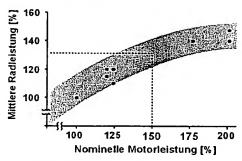


= 2,5.Ordnung = 5,0.Ordnung = 7,5.Ordnung = 4,5.Ordnung = 5,5.Ordnung

Figur 65

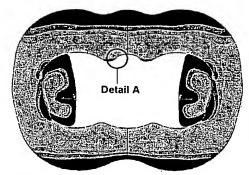


Figur 66

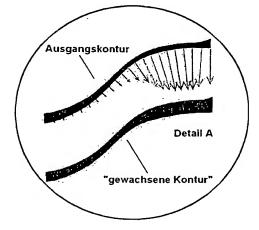


Kundenkollektive

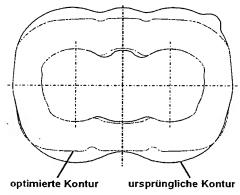
Figur 67



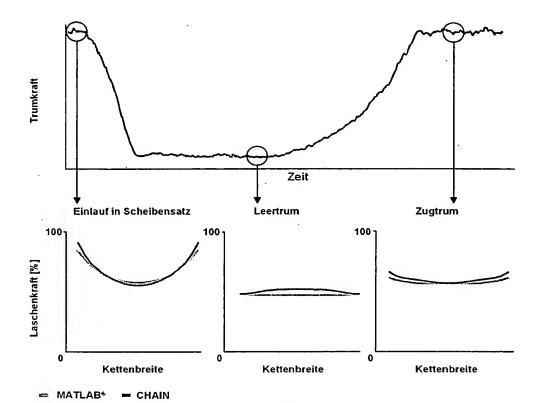
Figur 68



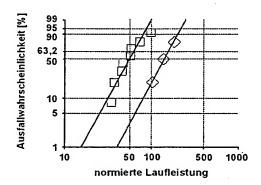
Figur 69



Figur 70



Figur 71

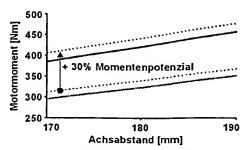


Herkömmliches StrickmusterOptimiertes Strickmuster

Figur 72

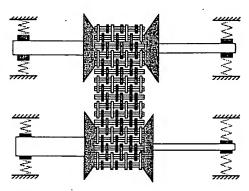




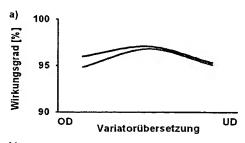


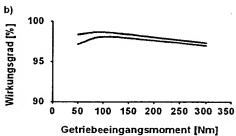
- ··· optimiertes System, Spreizung 5,5
- optimiertes System, Spreizung 6,0
- ··· Serienstand, Spreizung 5,5
- Serienstand, Spreizung 6,0

Figur 73



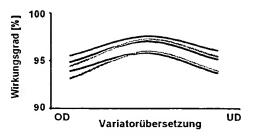
Figur 74





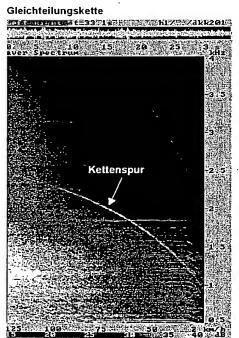
= 2000 min<sup>-1</sup> = 4000 min<sup>-1</sup>

Figur 75



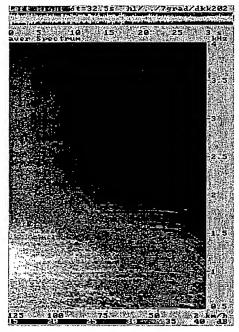
Variante		Schelbensatz- geometrie	Kette	Bemerkung
_	Basis A	ge- wölbt	37 mm	Serie multi- tronic <sup>®</sup>
REZER	В	11°	37 mm	-
_	С	7°	37 mm	-
-	D	ge- wölbt	37 mm "Leicht"	Welle: Ø + 2 mm
taan	E	ge- wölbt	37 mm	30% Über- anpressung

Figur 76



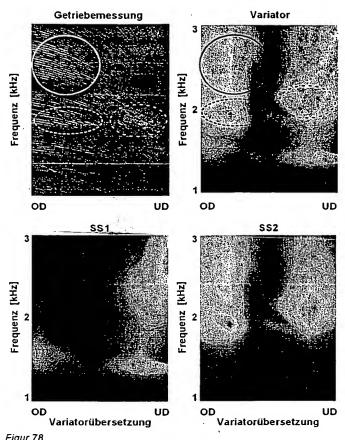
Figur 77

## Random-Kette

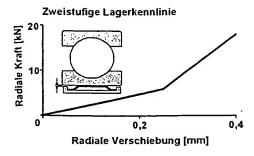


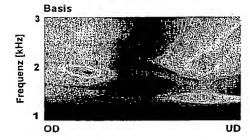




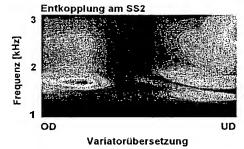


Figur 78





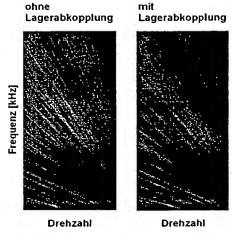




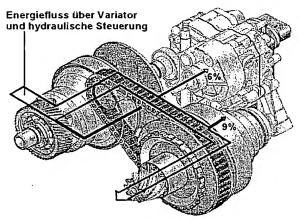
Figur 79







Figur 80



Figur 81





- Scheibensätze mit LuK Doppelkolben
- Kupplungskühlung mit Saugstrahlpumpe
- Geringe Leckage
  - Pumpe mit axialer und radialer Spaltkompensation
  - geringe Schieberzahl
  - enge Spiele und kleine Fertigungstoleranzen
  - geringe Anzahl von E-Ventilen

Figur 82



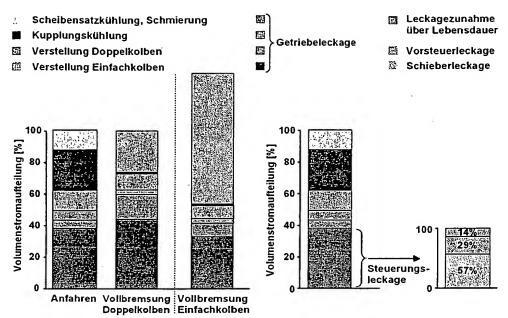
Hydraulische Verluste

Mechanische Verluste

- kleine Anpress-Sicherheits-faktoren (Schlupfregelung)
- Systemdruck abhängig vom aktuellen Motormoment
- geringe Rückstaudrücke

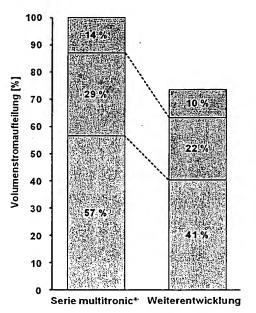






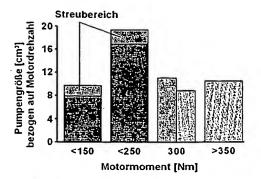






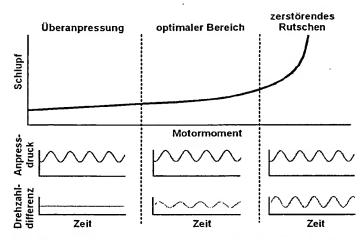
- Leckagezunahme über Lebensdauer
- ☑ Vorsteuerleckage
- Schieberleckage

Figur 84

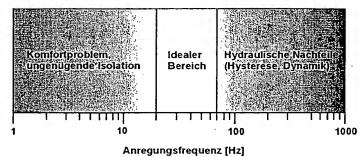


- konventionelle CVT
- ☐ Serie multitronic® mit LuK System
- 📆 LuK Weiterentwicklung

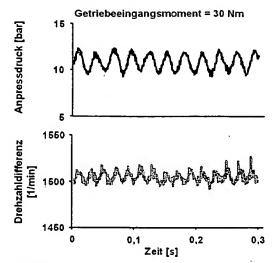
Figur 85

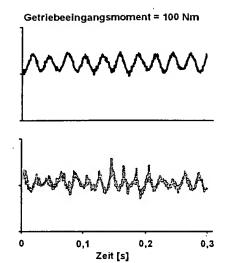


Figur 86



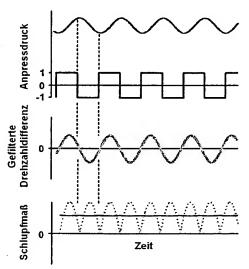
Figur 87



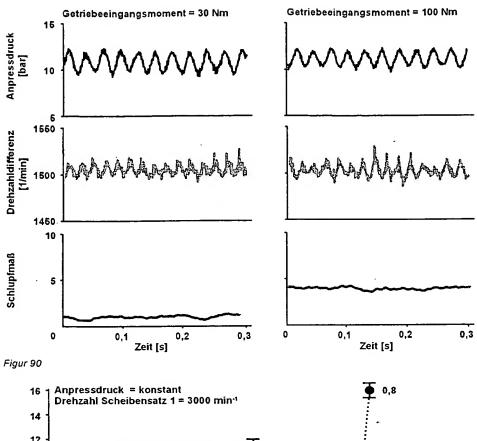


Figur 88





Figur 89



Anpressdruck = konstant
Drehzahl Scheibensatz 1 = 3000 min-1

Variatorübersetzung = 1,9

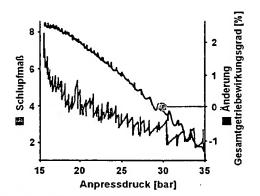
Variatorübersetzung = 1,9

Motormoment [Nm]

Anpressung multitronic\* mit zweistufigem Drehmomentfühler

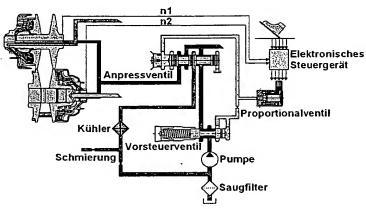
Figur 91

Variatorübersetzung = 0,47 Getriebeeingangsmoment = 310 Nm Drehzahl Scheibensatz 1 = 2000 min<sup>-1</sup>

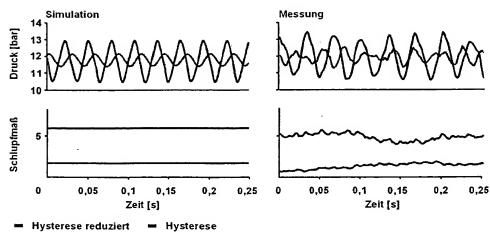


# Anpressung multitronic\* mit zweistufigem Drehmomentfühler

Figur 92

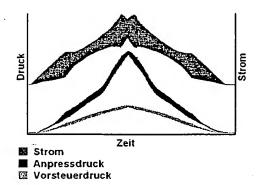


Figur 93



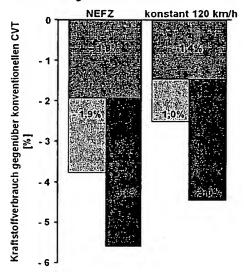
Trysterese reduziere — trysteres

Figur 94



Figur 95

Motormoment = 300 Nm Motorleistung = 160 kW



LuK Doppelkolben

kleinere Pumpe

LuK Drehmomentfühler

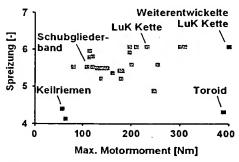
geringerer Anpressdruck

> multitronic\*

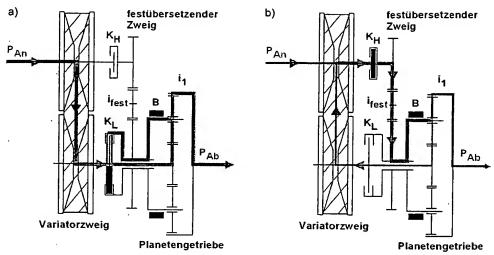
LuK Weiterentwicklung

Schlupfregelung und optimierte Hydraulik

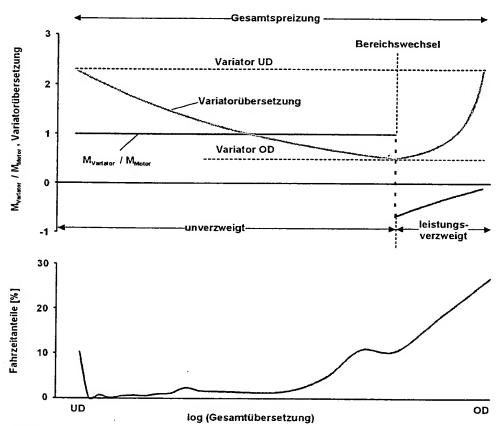
Figur 96



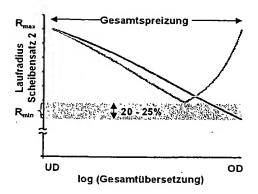
Figur 97



Figur 98

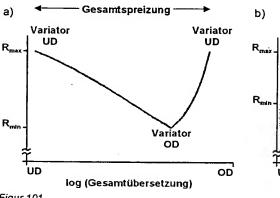


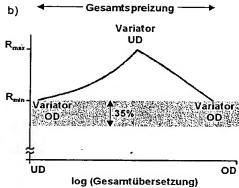
Figur 99



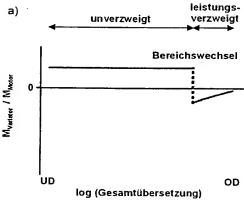
- unverzweigter Triebstrang
- □ Zweibereich-CVT

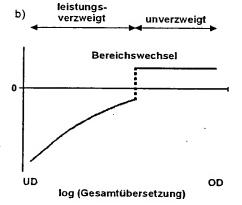
Figur 100





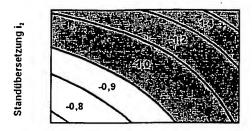
Figur 101





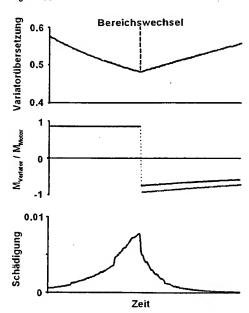
Figur 102

 zu hohe Variatormomente am Bereichswechsel

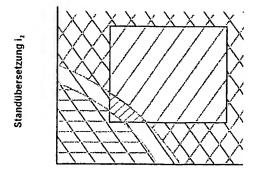


Standübersetzung i,

Figur 103



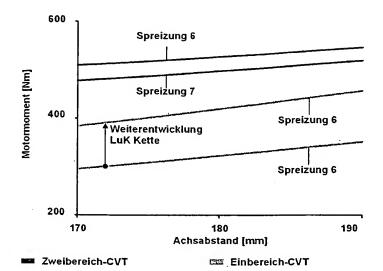
Figur 104



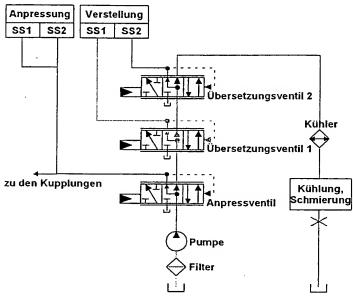
Standübersetzung i,

### Ausschlusskriterien:

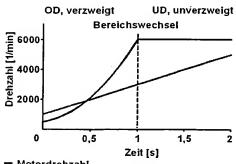
- R-Übersetzung, Variatormoment
- Bauraumbedingungen
- M Spreizung



Figur 106

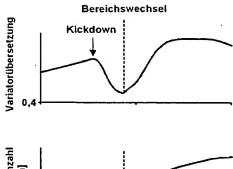


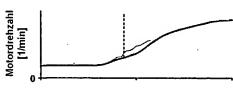
Figur 107

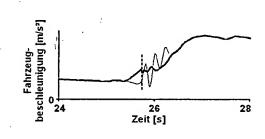


MotordrehzahlDrehzahl Scheibensatz 2

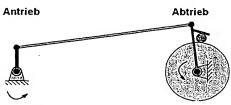
Figur 108



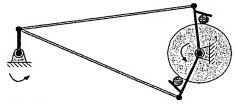




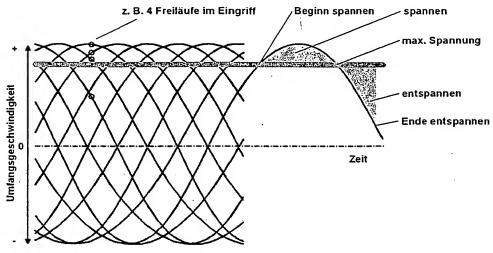
Figur 109



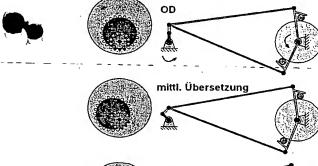
Figur 110

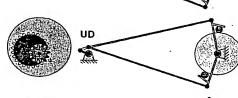


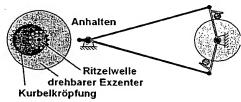
Figur 111



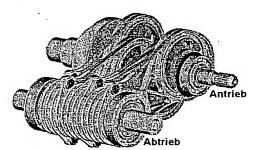
Figur 112



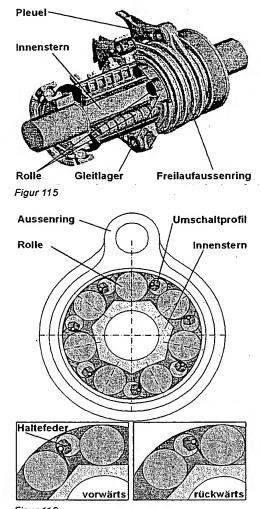




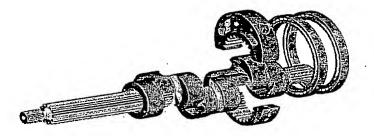
Figur 113

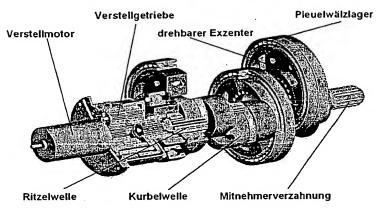


Figur 114

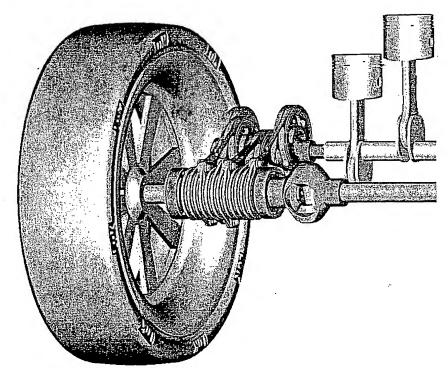


Figur 116

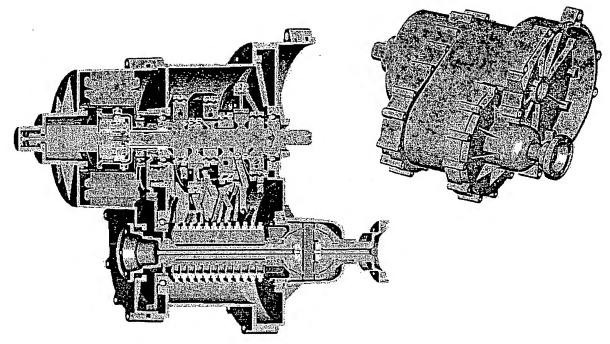




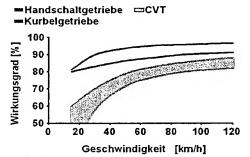
Figur 117



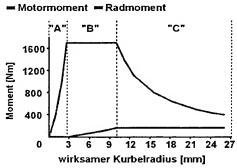
Figur 118



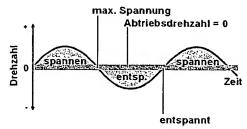
Figur 119



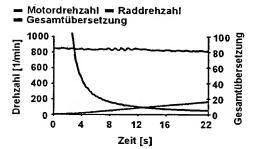
Figur 120



Figur 121

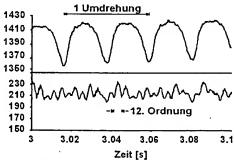


Figur 122

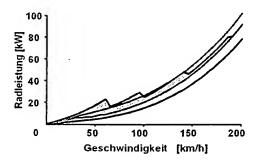


Figur 123

- Motordrehzahl [1/min]
- Abtriebsdrehzahl [1/min]

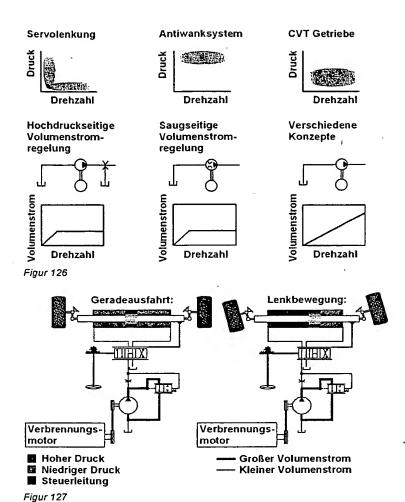


Figur 124

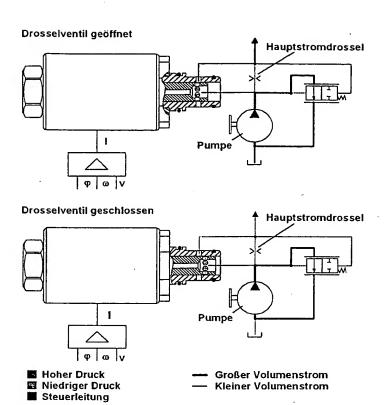


- mit Motorbremse
- mit Kurbelgetriebe
  Luft- und Rollwiderstand

Figur 125

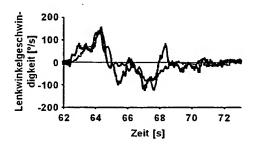


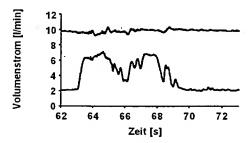


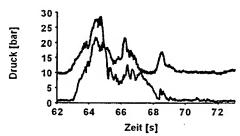






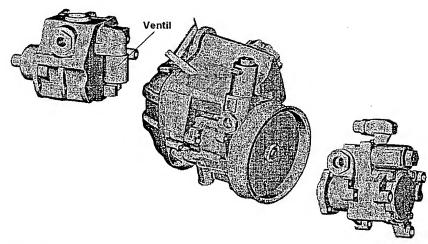




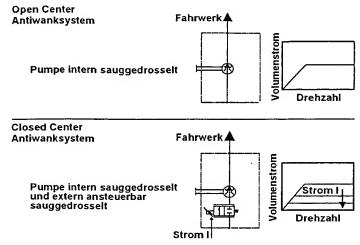


Standardpumpe
EV<sup>2</sup>

Figur 129



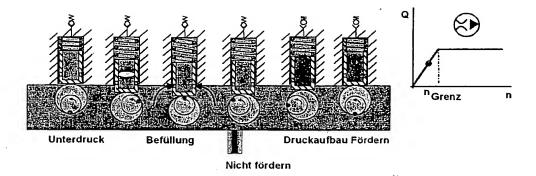
Figur 130

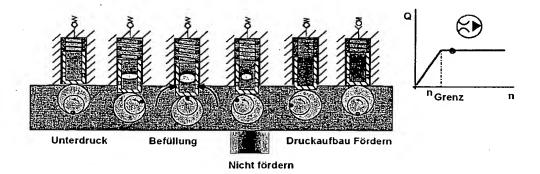


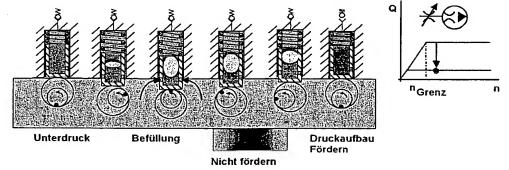
Figur 131



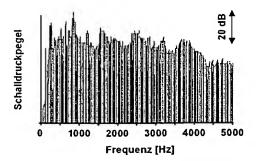






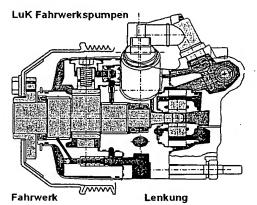


Figur 132



- Ausgangszustand
- □ Optimiertes Design

Figur 133



 $V_{geo} = 6 - 6,7 \text{ cm}^3$ 

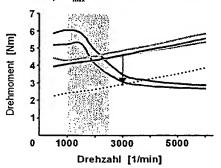
p<sub>max</sub> = 200 bar

 $V_{geo} = 8,4 - 15 \text{ cm}^3$ 

 $p_{max} = 135 \text{ bar}$ 

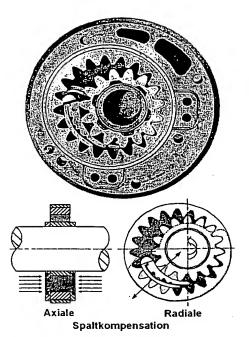
Figur 134

n = 1000 min<sup>-1</sup>, Q = 10,5 l/min, p = 20 bar, T = 90°C,  $Q_{max}$  erforderlich = 15 l/min

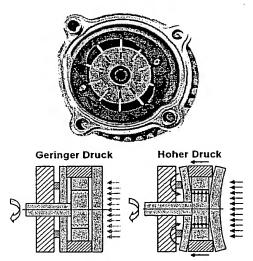


- = Sauggedrosselte Radialkolbenpumpe
- Verstellbare Flügelzellenpumpe
- Kompensierte Innenzahnradpumpe
- **≖** Konstant-Flügelzellenpumpe
- Zweiflutige FZP zweiflutiger Betrieb 100%
- ··· Zweiflutige FZP einflutiger Betrieb 50%

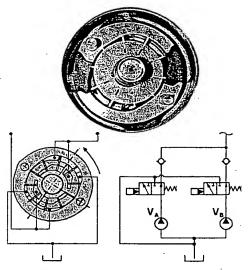
Figur 135



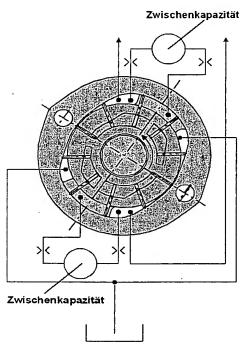
Figur 136



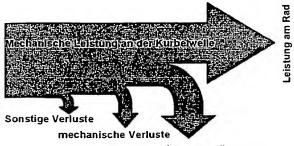
Figur 137



Figur 138



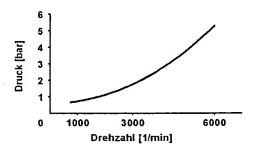
Figur 139



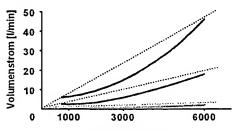
Nebenaggregate (~3% für die Ölpumpe)



Figur 140



Figur 141

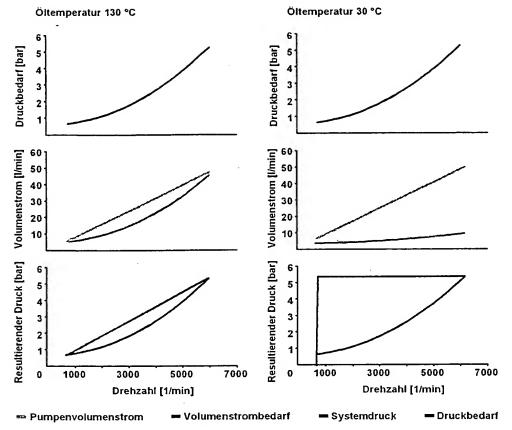


Drehzahl [1/min]

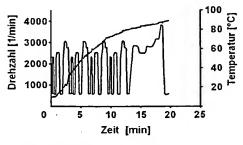
··· Konstantpumpe = Motorbedarf 130 °C ····Konstantpumpe = Motorbedarf 90 °C

····Konstantpumpe 🚥 Motorbedarf 30 °C

Figur 142

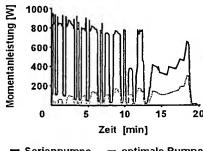


Figur 143



- Öltemperatur - Drehzahl

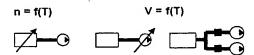
Figur 144



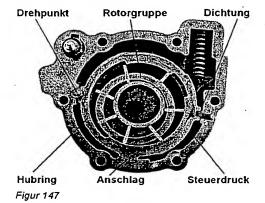
Serienpumpe soptimale Pumpe

Figur 145





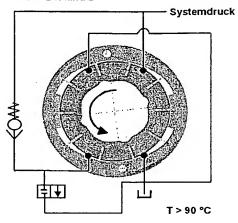
Figur 146





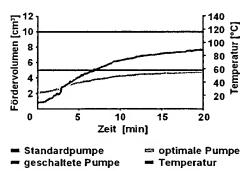
# Schaltzustand 1 Systemdruck T≤90 °C

### Schaltzustand 2

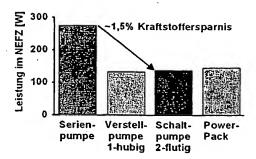


- Niederdruck - Hochdruck

Figur 148



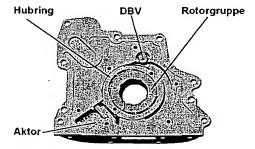
Figur 149



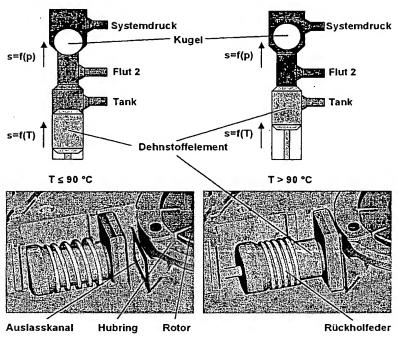
Figur 150

Lösungsprinzip	Variabler E-Motor	Variabler Hub	Schalt- pumpe
Annäherung an den Druckbedarf			
Leistungsbedarf			
Betriebssicherheit			
Bauraum	0		
Kosten			

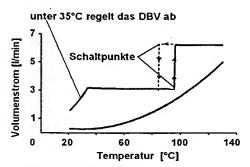
Figur 151



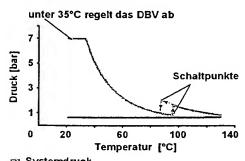
Figur 152



Figur 153

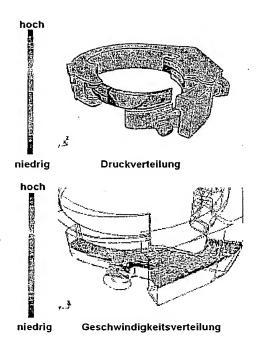


SystemvolumenstromVolumenstrombedarf



Systemdruck
Druckbedarf

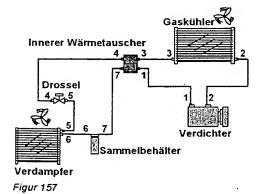
Figur 154

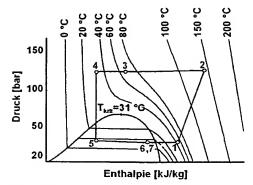


Figur 155

Name	HCFC (R12)	HFC (R134a)	CO <sub>2</sub> (R744)
Ozonschädigung	ja	ja	keine, da natürliches Gas
Treibhauspotenzial	GWP = 8100	GWP = 1300	GWP = 1
CO <sub>2</sub> Ausstoß durch Betrieb	2600 kg / PKW	2600 kg / PKW	1800 kg / PKW
CO <sub>2</sub> GWP-Äquivalent über Lebensdauer	8100 kg / PKW	1300 kg / PKW	0,5 kg / PKW
Summe	10700 kg / PKW	3900 kg / PKW	1800,5 kg / PKW

Figur 156

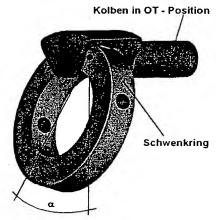




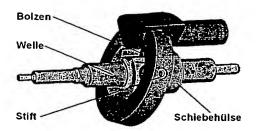
Figur 158

Bauart	Flügelzellenverdichter	Spiralverdichter	Axialkolbenverdichter
Massenstrom- regelung	Bypass	Bypass	Hubverstellung über Schwenkmechanis- mus
Wirkungsgrad der Kli- maanlage, Verdichter mit max. Massen- strom (COP)	(2)	(2)	(2)
Wirkungsgrad der Kli- maanlage, Verdichter mit abgeregeltem Massenstrom (COP)	(1)	(1)	(1,8)
Geräusch			
Kosten	<b>®</b>		

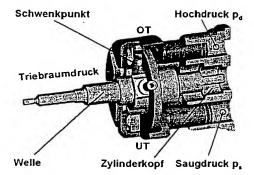
Figur 159



Figur 160

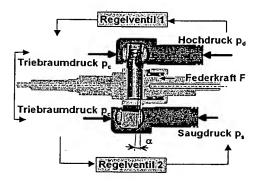


Figur 161



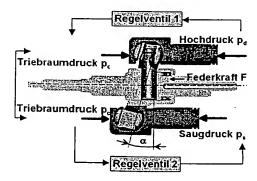
Figur 162

# Schwenkwinkel $\alpha$ bei $p_c > p_s$



Figur 163

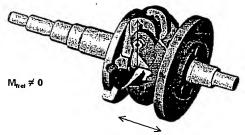
## Schwenkwinkel $\alpha$ bei $p_c = p_s$



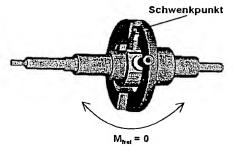
Figur 164

Schwenk- mechanismus	Maximal- drehzahl	Regelbarkeit bei CO <sub>2</sub>	Geräusch
Taumelscheibe			
Schwenkscheibe			
Schwenkring			

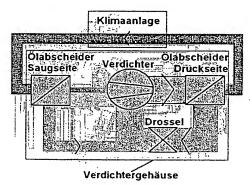
Figur 165



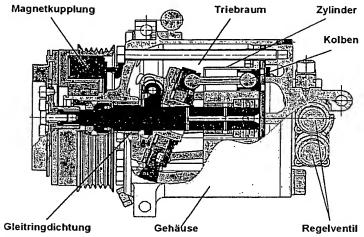
Figur 166



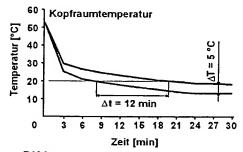
Figur 167



Figur 168



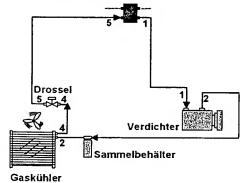
Figur 169

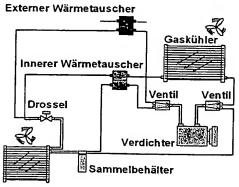


= R134 a = CO<sub>2</sub>

Figur 170

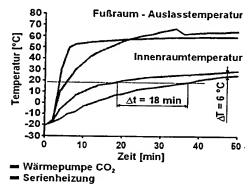
### Externer Wärmetauscher





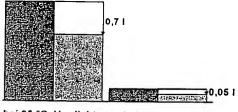
Verdampfer / Gaskühler

Figur 172



Figur 173

Klimaanlage "an" Klimaanlage "aus"



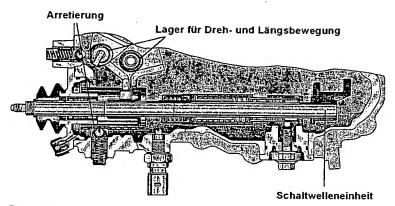
bei 25 °C, Verdichter mit Kupplung

bei 25 °C, Verdichter ohne Kupplung (Verdichter läuft mit)

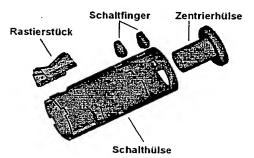
■ R134 a 図 CO<sub>2</sub>

Figur 174

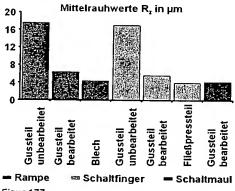
0

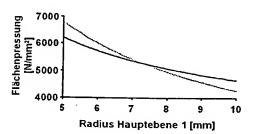


Figur 175



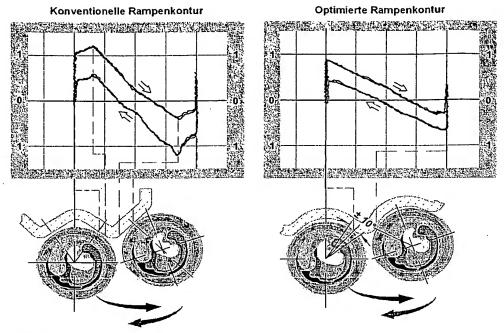
Figur 176





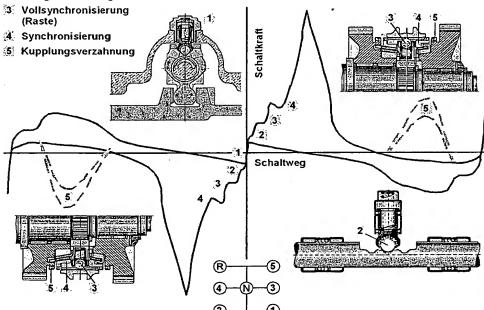
- max. Vergleichsspannung nach Mises [N/mm²]
- max. Flächenanpressung [N/mm²]

Figur 178

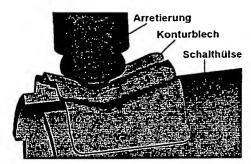


Figur 179

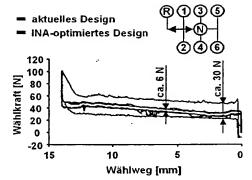
- (1) Neutralrastierung
- 2 Stangenrastierung



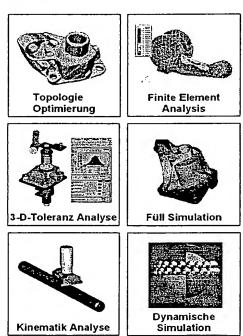
Figur 180



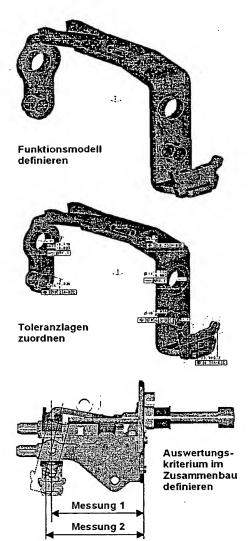
Figur 181



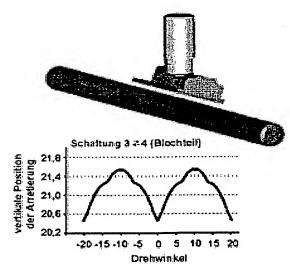
Figur 182



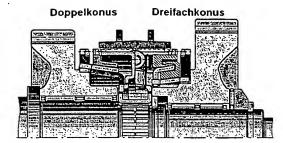
Figur 183



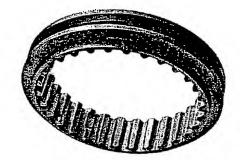
Figur 184



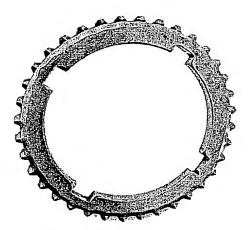
Figur 185



Figur 186



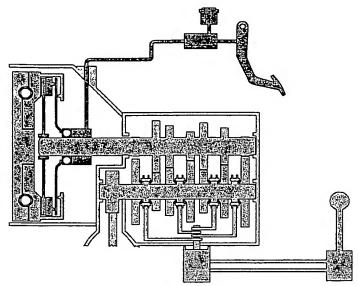
Figur 187



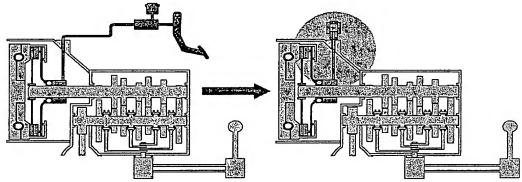
Figur 188

- Vorsynchronisation
- Außensynchronisation
- zwischenring
- Innensynchronisation
- Kupplungsverzahnungsscheibe
- Radkonusring
- Schiebemuffe
- 🚥 Synchronkörper

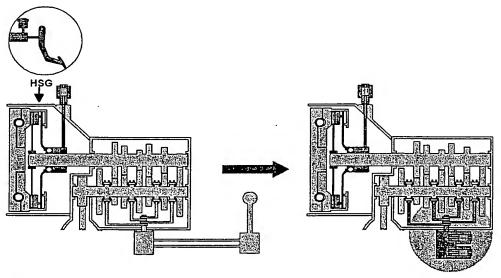




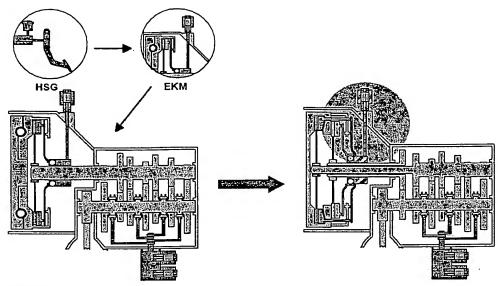
Figur 189



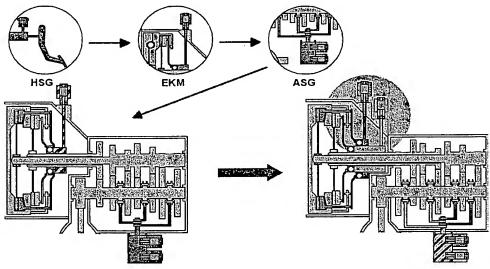
Figur 190



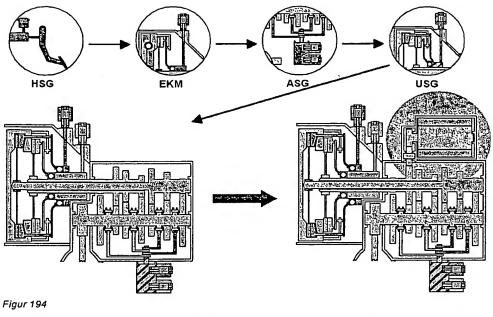
Figur 191

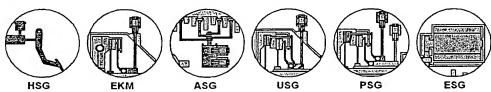


Figur 192



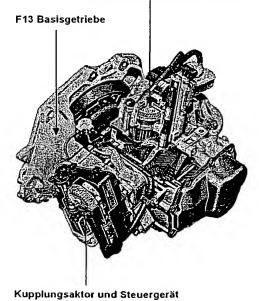
Figur 193





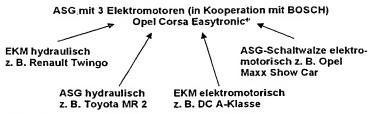
Figur 195

## Getriebeaktor (2 Elektromotoren)

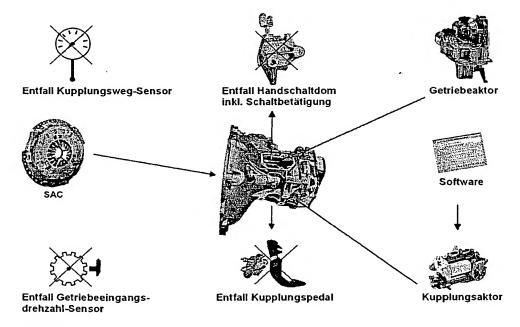


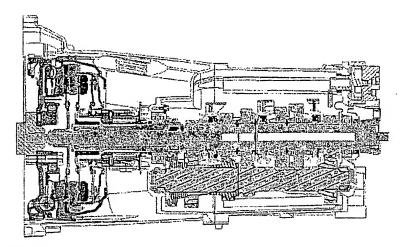
Figur 196



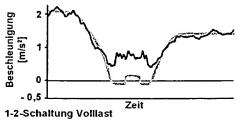


Figur 197



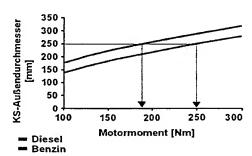


Figur 199



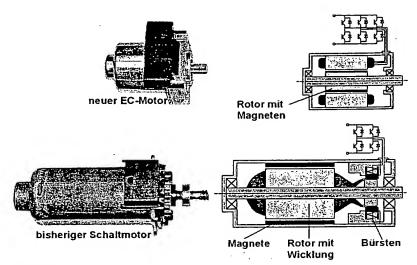
- Fahrzeugbeschleunigung ASG
  Fahrzeugbeschleunigung USG (Messung)

Figur 200

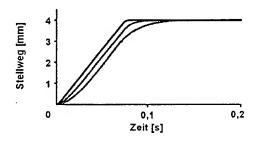


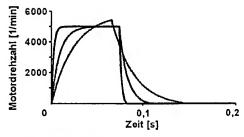
Figur 201





Figur 202





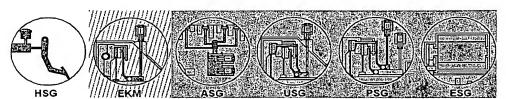
- neuer Motor (EC)
- heutiger Schaltmotor (DC)
- heutiger Kupplungsmotor (DC)

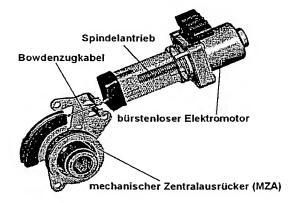
Figur 203

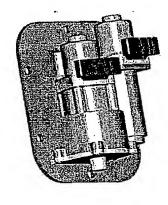


	heutiger Kupplur motor (DC)	ıgs-	heutiger Schaltm (DC)	otor	neuer Motor (EC)	
Leistungsdichte	101 W/kg		163 W/kg		267 W/kg	
		100%		162%		266%
Massenträgheit	30,4 · 10 <sup>-6</sup> kgm <sup>2</sup>		25,0 · 10 <sup>-6</sup> kgm <sup>2</sup>		6,5 · 10 <sup>-6</sup> kgm <sup>2</sup>	
		100%		82%		21%
mechanische Zeitkonstante	27,75 ms		7,46 ms		1,88 ms	
		100%		27%		6,8%
Gewicht	693 g		813 g		438 g	
		100%		117%		63%
Volumen	166 cm <sup>2</sup>		162 cm <sup>2</sup>	·····	62 cm <sup>2</sup>	
		100%		98%		37%

Figur 204

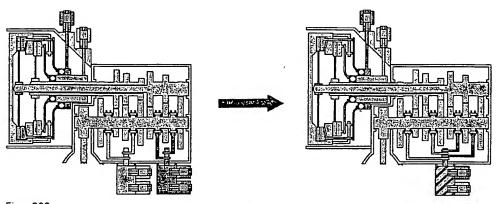




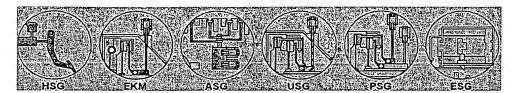


Kupplungsaktor

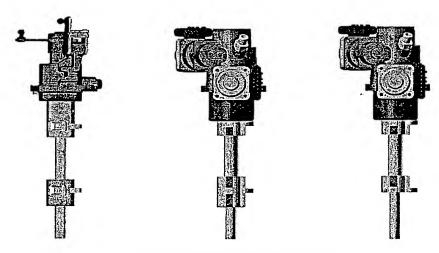
Getriebeaktor



Figur 206









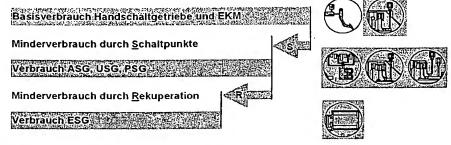


Figur 207

#### Verbrauch Automatikgetriebe

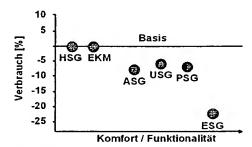


#### Verbrauch XSG-Familie

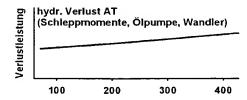




Figur 208



Figur 209



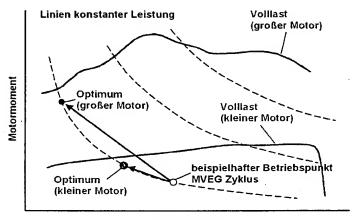


Relativer Mehrverbrauch AT (hydr. Verluste, Schaltpunkte wie HSG)

100 200 300 400

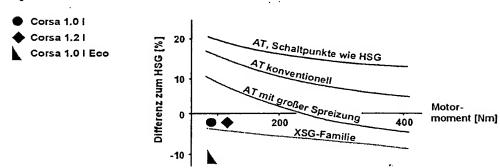
Motormoment [Nm]

Figur 210

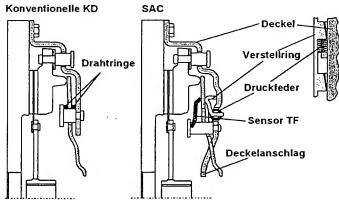


Motordrehzahl

Figur 211



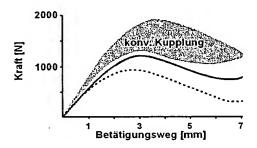
Figur 212



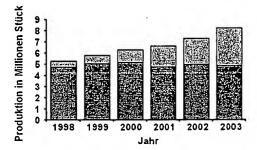




# SAC I für Fußkraftbetätigung SAC I für automatische Kupplung

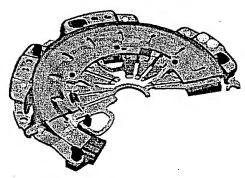


Figur 214

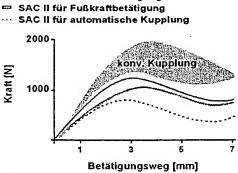


Konventionelle KD

Figur 215

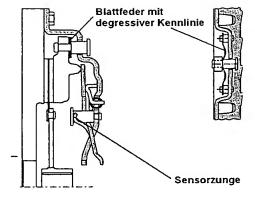


Figur 216



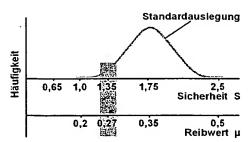
■ SAC I für Fußkraftbetätigung

Figur 217

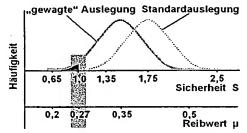






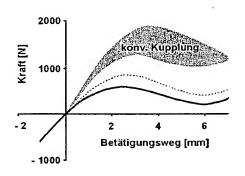


Figur 218



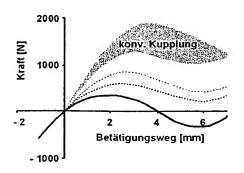
Figur 219

- · · · SAC II
- SAC II mit 30% reduziertem Übertragungsmoment

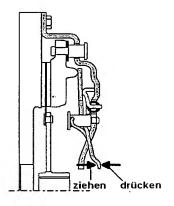


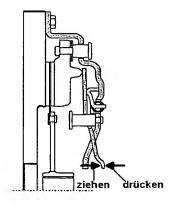
Figur 220

- · · · SAC II
- ··· SAC II mit 30% reduziertem Übertragungsmoment
- PPP Kupplung

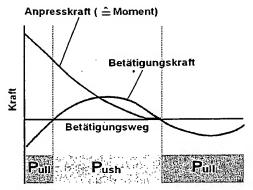


Figur 221

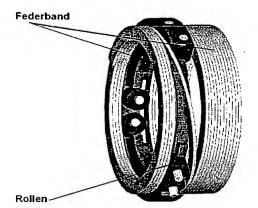




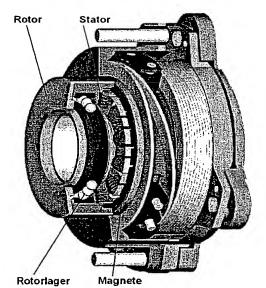




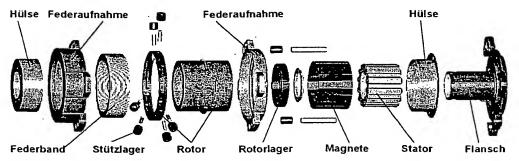
Figur 222



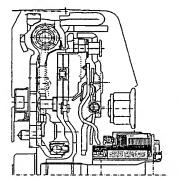
Figur 223

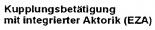


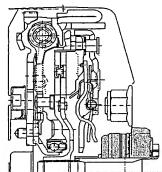
Figur 224



Figur 225

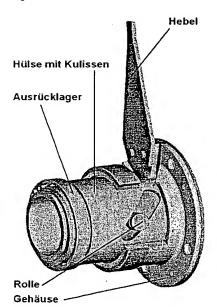






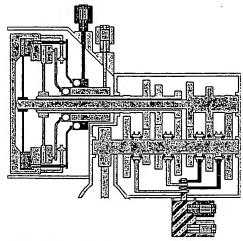
Serienstand mit CSC

Figur 226



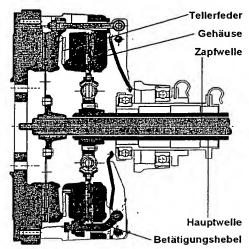


Figur 227

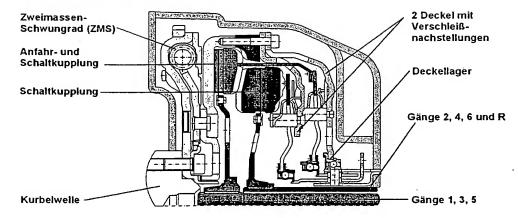


Figur 228

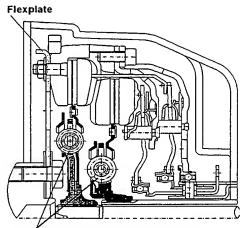




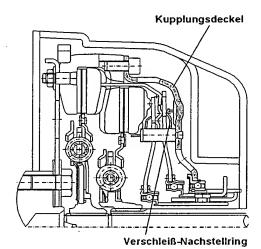
Figur 229



Figur 230

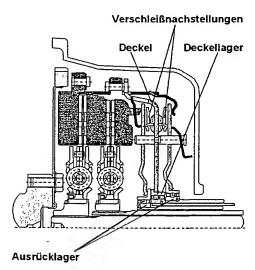


Torsionsdämpfer für Schlupfregelung

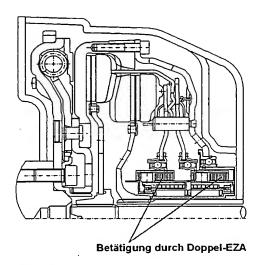


Figur 232

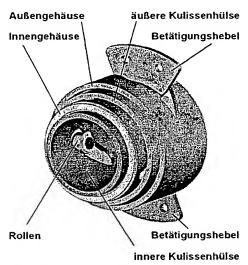




Figur 233

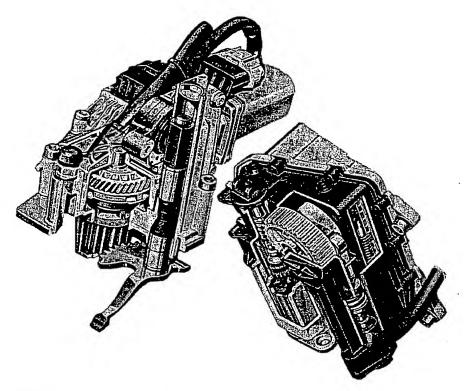


Figur 234

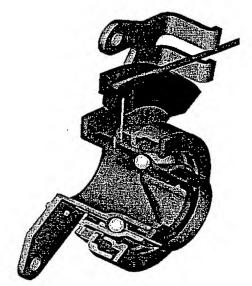


Figur 235

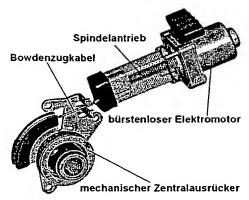




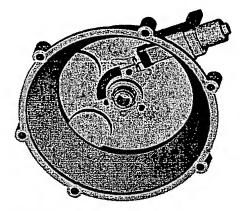
Figur 236



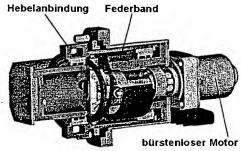
Figur 237



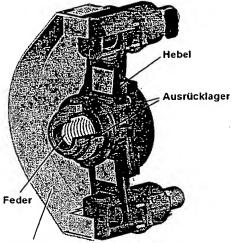
Figur 238



Figur 239

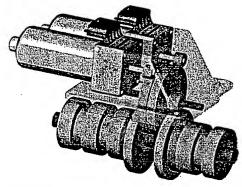


Figur 240

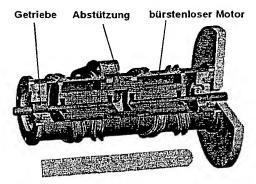


Kupplungsglocke

Figur 241



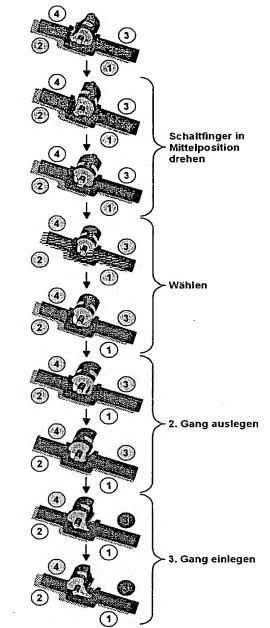
Figur 242



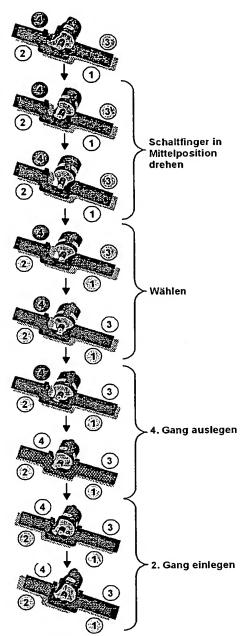
Figur 243



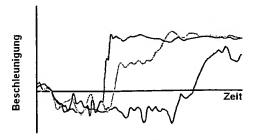




Figur 244

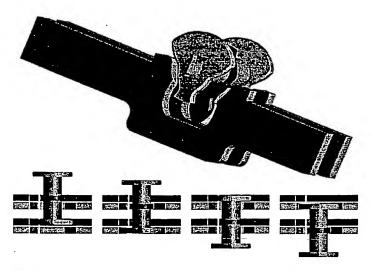


Figur 245



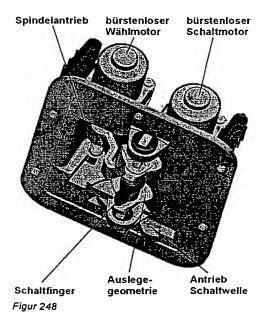
- LuK ASG mit Active Interlock 5 → 2
  LuK ASG 5 → 2
  Schaltwalze 5 → 3

Figur 246



- 🖾 Schaltschienen gerade Gänge
- Schaltschienen ungerade Gänge
- Schaltfinger
- Auslegegeometrien





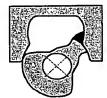








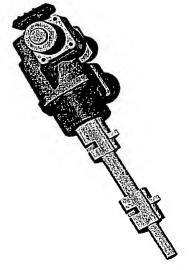
Auslegen 1



Sperren

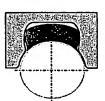




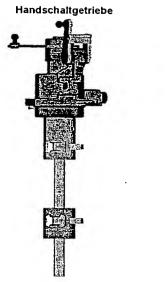


Figur 250

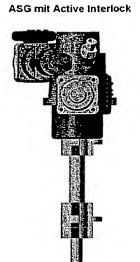




Figur 251

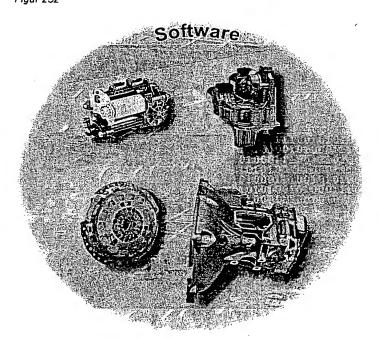


Figur 252

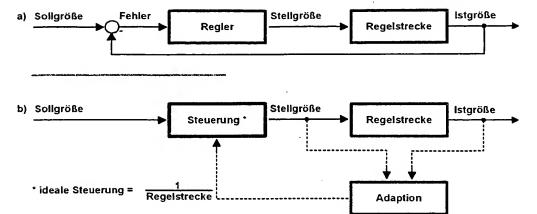


## PSG mit Active Interlock

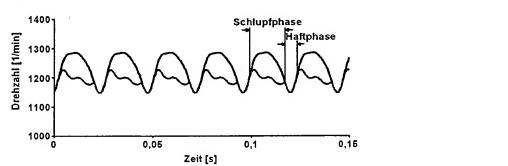




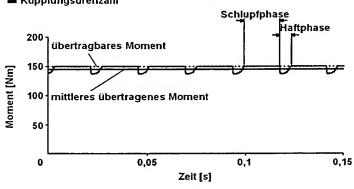
Figur 253



Figur 254

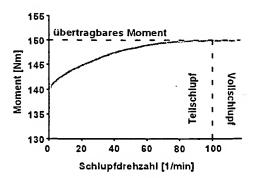


## MotordrehzahlKupplungsdrehzahl



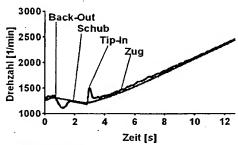
- übertragenes Moment

Figur 255

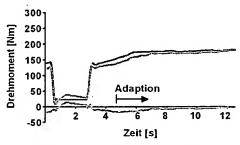


mittleres übertragenes Moment bei konstanter Anpresskraft

Figur 256



MotordrehzahlGetriebeeingangsdrehzahl



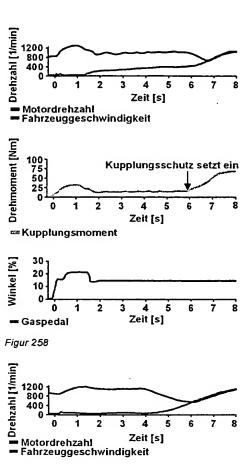
📨 Gesamtmoment 🖛 Steuerungsanteil

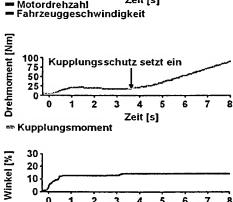
Regelungsanteil

Figur 257







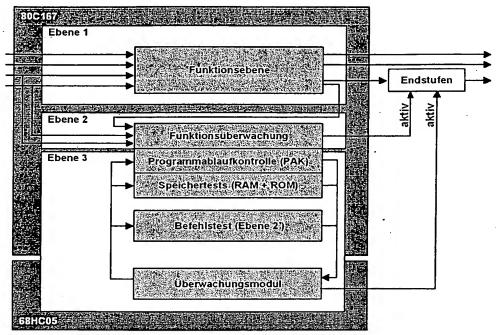


2 3 4 5 Zeit [s]

**-** Gaspedal Figur 259

		Situation				
		Geschlossen	Zwischen- stellung	Geöffnet		
Reaktion	Schließen	<b>Maga</b>	?	4		
	Öffnen	4	?			
	Stehenbleiben	Sept.	V.	<b>W</b>		

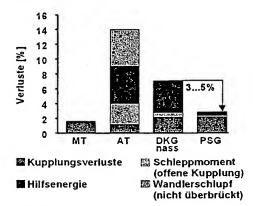
Figur 260



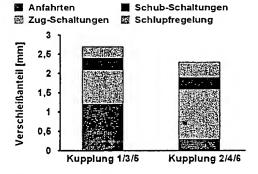
Figur 261

	Trocken- kupplung	Nass- kupplung
Verbrauch		
Überlastbarkeit Berganfahrten		
Option für modulare Familien		
Bauraum/ Gewicht		
Verhalten bei Ausfali		
Schaltqualität Regelbarkeit		

Figur 262



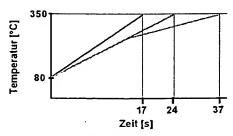
Figur 263



Fahrzeug Motor Kupplungen Lebensdauer 2000 kg 400 Nm Durchmesser 250/220 240 000 km

Figur 264



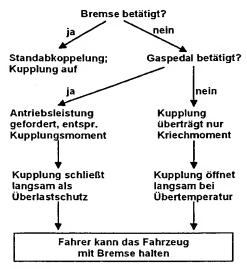


Fall 1: 1. Gang wie Handschalter

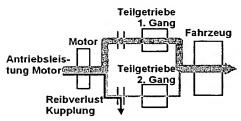
Fall 2: 1. Gang 20% kürzer

Fall 3: Anfahren über beide Kupplungen

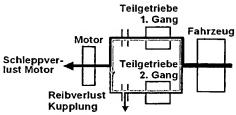
Figur 265



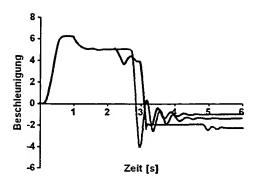
Figur 266



Figur 267

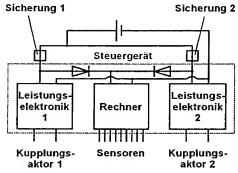


Figur 268

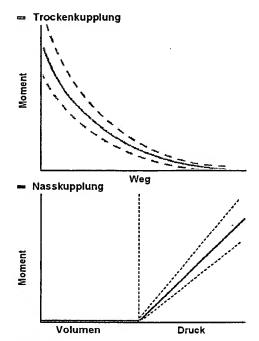


- 1. Gang: Zug -> Schub-Wechsel
- Ausfall bei 1 → 2 Schaltung, danach Zug-Schub-Wechsel
- Ausfall bei 1 → 2 Schaltung, danach Zug → M<sub>mot</sub> = 0 Nm

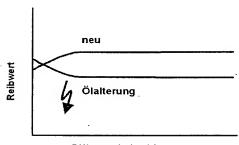
Figur 269



Figur 270

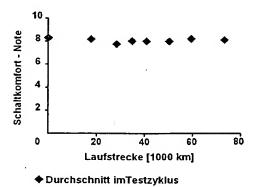


Figur 271



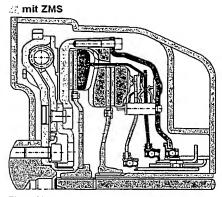
Differenzdrehzahl

Figur 272

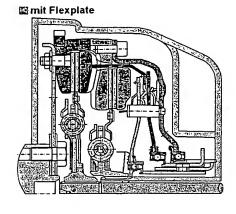


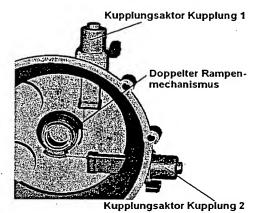
•

Figur 273



Figur 274

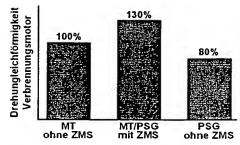




Figur 275

	ohne ZMS	mit ZMS
MT / ASG	0,2 kgm <sup>2</sup>	0,25 kgm <sup>2</sup>
PSG	0,25 kgm <sup>2</sup>	0,35 kgm <sup>2</sup>

Figur 276

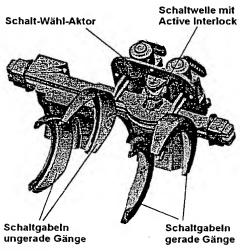


Figur 277

	Mehrverbrauch	
	Manueller Modus	Automatik Modus
Optimierung Motorbetriebspunkt	± 0%	-5,0%
Reduktion Massenträgheit	-0,5%	-0,5%
Schlupfregelung	+0,5%	+1,0%
Bilanz	± 0%	-4,5%

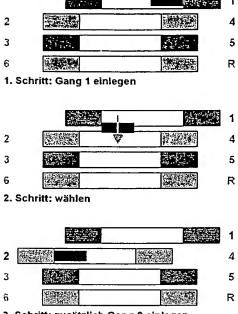
beispielhafter Durchschnittswert

Figur 278



Figur 279

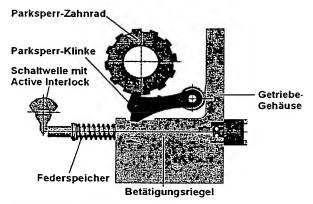




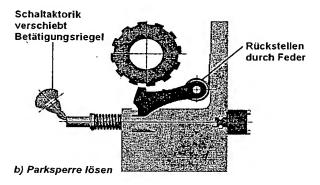
3. Schritt: zusätzlich Gang 2 einlegen

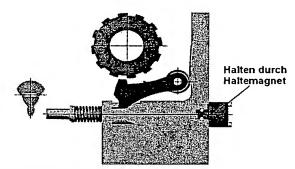
Figur 280





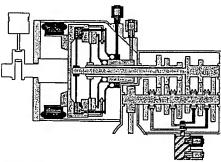
#### a) Parksperre betätigt



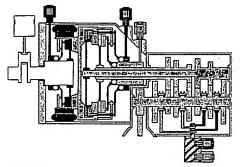


### c) Parksperre offen halten

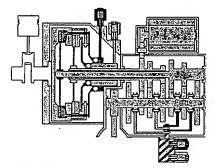
Figur 281



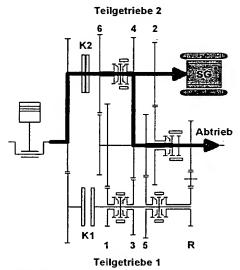
Figur 282



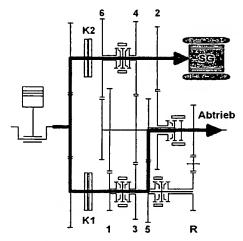
Figur 283



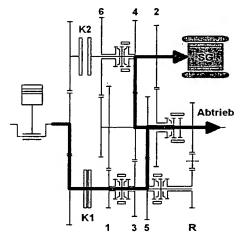
Figur 284



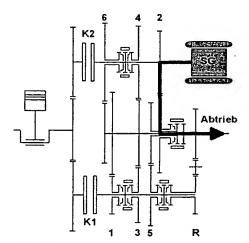
Figur 285



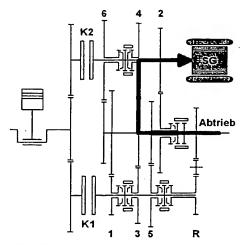
Figur 286



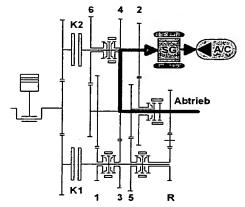
Figur 287



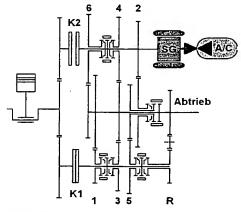
Figur 288



Figur 289

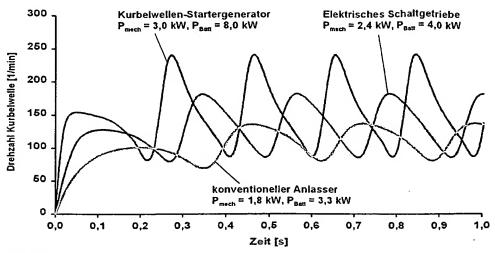


Figur 290

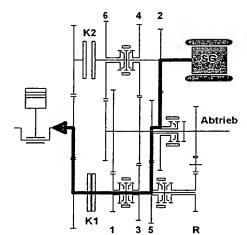


Figur 291



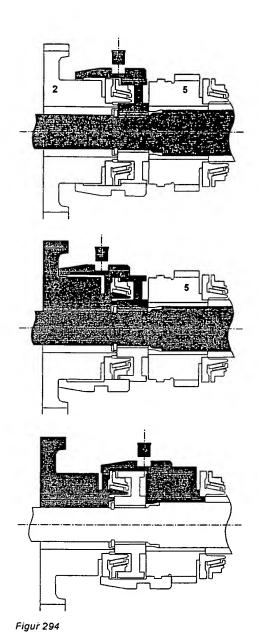


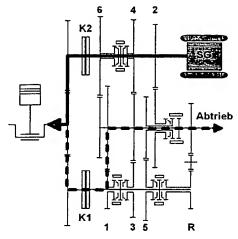
Figur 292



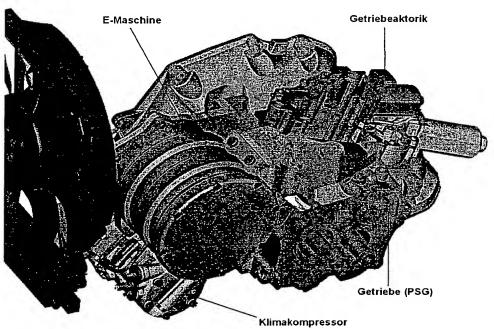
Figur 293



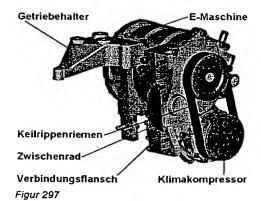




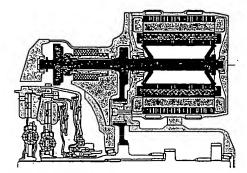
Figur 295



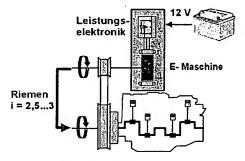






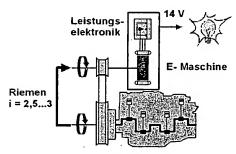


Figur 298



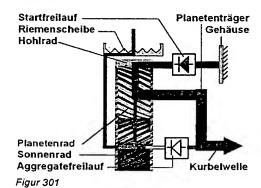
ohne Getriebe i = 1 mit Getriebe i = 2,5...3,5

Figur 299

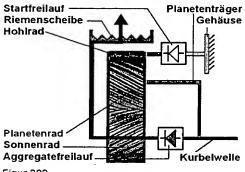


i = 1 mit und ohne Getriebe

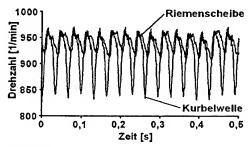
Figur 300



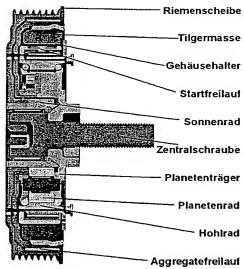




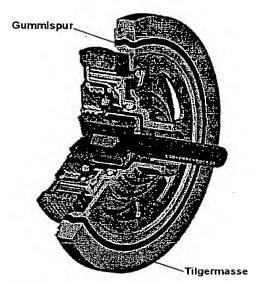
Figur 302



Figur 303



Figur 304



Figur 305





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